



TMMOB İMO İSTANBUL ŞUBESİ
2017 YILI (ŞB-1) MESLEK İÇİ EĞİTİM SEMİNERLERİ

CAMIN YAPILARDA KULLANIMI VE HESAP ESASLARI

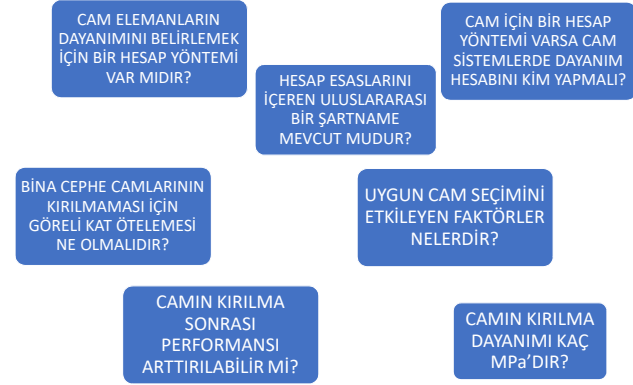
04.05.2017 Karaköy



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CAMIN YAPILARDA KULLANIMI VE HESAP ESASLARI



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GÖRSEL

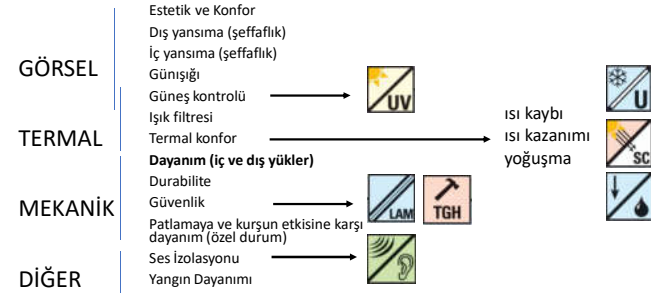
YERDEN ONLARCA METRE YÜKSEKLİKTE CAM DÖŞEMELİ BİR KÖPRÜDE NE KADAR GÜVENDE OLABİLİRSİNİZ?

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CAMIN YAPILARDA KULLANIMI

Cam seçimini etkileyen faktörler

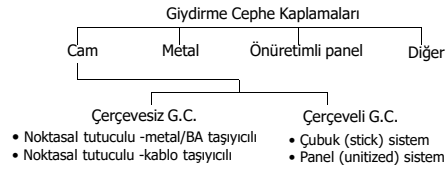


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CAMIN YAPILARDA KULLANIMI

KAPLAMA ELEMANI



TAŞIYICI ELEMAN

- kiriş, kolon, plak (plate), levha (membrane), kabuk (shell)

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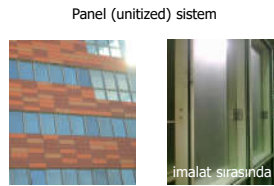
DOLGU ELEMANI



CAMIN YAPILARDA KULLANIMI

DOLGU ELEMANI

Çerçevesiz G.C.

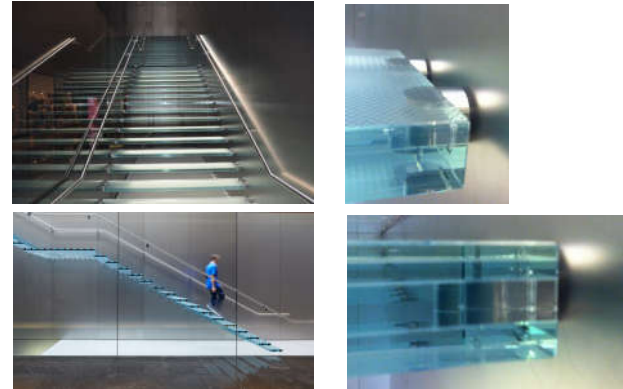


Çerçevesiz G.C.



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TAŞIYICI ELEMAN



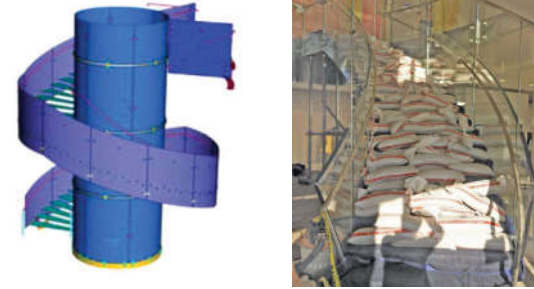
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1. MALZEME VE ÜRÜN ÖZELLİKLERİ
2. TEMEL TASARIM İLKELERİ
3. İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI
4. BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

HESAP ESASLARI

1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Soda-kireç-silikat camı tipik karışımı (TS EN 572-1)

*[1]

Malzeme	Formül	Karışım Yüzdesi
Silikat	SiO ₂	69-74%
Kireç	CaO	5-14%
Soda	Na ₂ O	10-16%
Magnezyum oksit	MgO	0-6%
Alüminyum oksit	Al ₂ O ₃	0-3%

EN 572-2 Yüzdürme (float) cam

EN 572-3 Parlatılmış telli cam

EN 572-4 Çekme düz cam

EN 572-5 Desenli cam

EN 572-6 Desenli telli cam

EN 572-7 Telli veya telsiz kanal şekilli cam

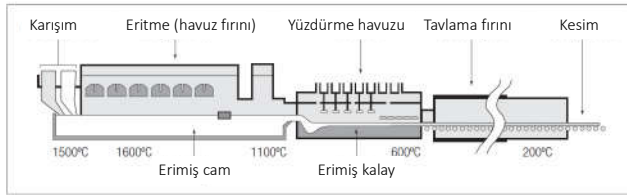
EN 572-8 Piyasaya arz boyutları ve son kesim boyutları

EN 572-9 Uygunluk değerlendirilmesi/mamul standardı

HESAP ESASLARI

1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Günümüzde kullanılan cam biçimlendirme yöntemlerinden float (yüzdürme) yöntemi



Nominal kalınlık: 2, 3, 4, 5, 6, 8, 10, 12, 15, 19 (25*) mm
Standard plaka boyutu: 6 m x 3.21 m
Standard dışı plaka 25m (Çin)

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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

GÖRSEL

FLOAT CAM ÜRETİMİ

HESAP ESASLARI

1-MALZEME VE ÜRÜN ÖZELLİKLERİ

İmalat Toleransları TS EN 12150-1

*[2]

- boy 2 – 4 mm
- gönye 2 – 4 mm
- float cam kalınlığı – tabloya bakınız
- delik çapı 0,5 mm
- delik konumu 0,5 – 2 mm

Nominal kalınlık (mm)	Tolerans (mm)
2 - 6 mm	± 0,2 mm
8 - 12 mm	± 0,3 mm
15 mm	± 0,5 mm
19 - 25 mm	± 1,0 mm

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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

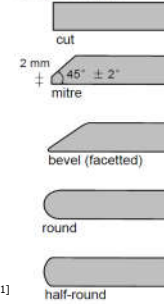
Kenar Kalitesi TS EN 12150-1

*[2]

KENAR KALİTESİ

- İşlenmemiş, kesim: keskin kenarlı
- Çapağı alınmış: keskin kenarları kırılmış ya da taşlama ile
- Kaba taşlanmış kenarlı
- Düzgün taşlanmış kenarlı
- Parlatılmış kenarlı

KENAR ŞEKLİ



Kenar işleminin dayanma etkisi için bkz. Maria Lindqvist PhD. 2013 [21]

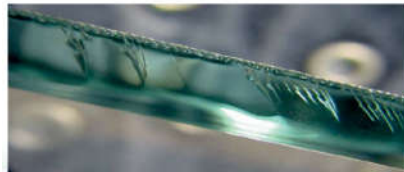
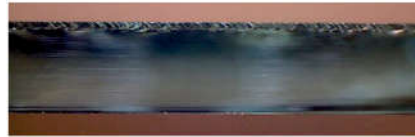
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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Kenar Kalitesi TS EN 12150-1

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- Parlatılmış kenarlı



HESAP ESASLARI

1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Kenar Kalitesi TS EN 12150-1

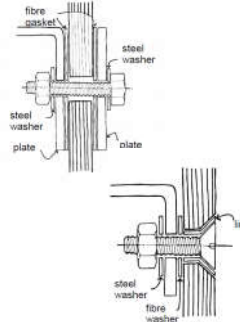
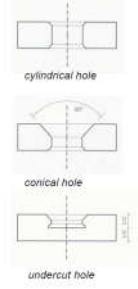


Şekil 6. Soğutmada kırılmaya yol açabilen hata örnekleri

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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Delik delme



water cutting

Fotoğraf Kaynağı: Sustainable Constructions under Natural Hazards and Catastrophic Events 520121-1-2011-1-CZ-ERA MUNDUS-EMMC

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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Delik delme (GENEL KURALLAR)

Delik çapı cam kalınlığından büyük olmalı

Delikten kenara temiz mesafe en az cam kalınlığının iki katı olmalı

Delikten köşeye temiz mesafe en az cam kalınlığının altı katı olmalı

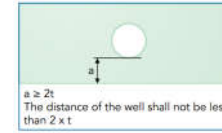


Fig. 7: Position of hole relative to edge

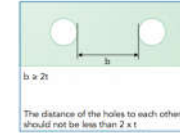


Fig. 8: Position of adjacent holes

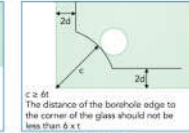


Fig. 9: Position of hole relative to corner

Fotoğraf Kaynağı: Guardian GlassTime, Standards, guidelines, tips

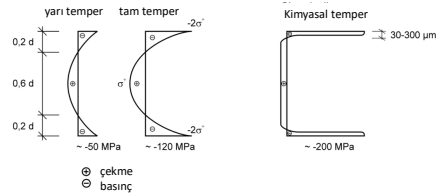
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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Camın Dayanımının Arttırılması

Eğilme dayanımlarına göre camların sınıflandırılması:

- tavllanmış cam (annealed) $f_b, k=45\text{Mpa}$
- yarı temper cam (half tempered, heat strengthened) $f_b, k=70\text{Mpa}$
- tam temper cam (fully tempered, thermally toughened) $f_b, k=120\text{Mpa}$
- kimyasal temperli cam (chemically strengthened) $f_b, k=150\text{Mpa}$



HESAP ESASLARI

1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Camın Dayanımının Arttırılması

GÖRSEL
KİMYASAL TEMPERLİ CAM DENEYİ-1

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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Camın Dayanımının Arttırılması

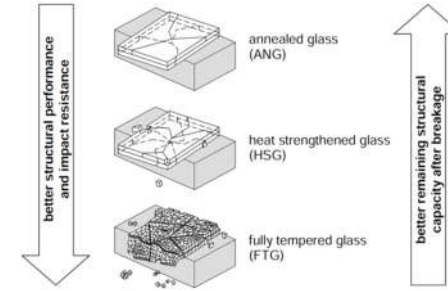
GÖRSEL
KİMYASAL TEMPERLİ CAM DENEYİ-2

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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Camların kırılma tipleri

*[3]

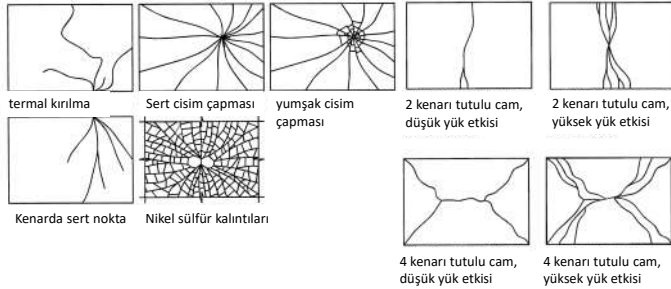


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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Camda Kırılma Sebepleri

*[3]

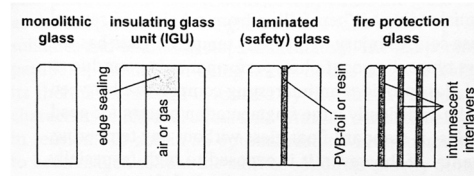


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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Cam Üniteleri

*[3]



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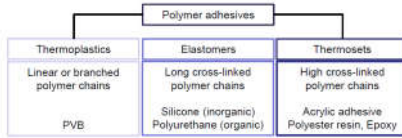
1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Ara katman tipleri

*[4]

Polyvinyl butyral (PVB)
Ionoplast Polymers
Ethylene Vinyl Acetate (Cross-Linked EVA)
Cast in Place (CIP) liquid resin
Thermoplastic polyurethane (TPU)

Thermoplastics: Relatively weak intermolecular forces hold molecules together in a thermoplastic. The material softens when exposed to heat, but returns to its original condition when cooled. Can be repeatedly softened by heating and then solidified by cooling, for improved performance. (linear and slightly branched polymers)
Elastomers: Rubbery polymers that can be stretched easily to several times their unstretched length and which rapidly return to their original dimensions. (cross-linked with low cross-link density)
Thermosets: Solidify or "set" irreversibly when heated and further heating cannot reshape the material. (3D networked polymers with high degree of cross-linking)



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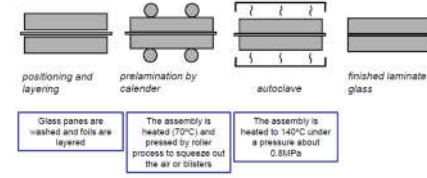
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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Ara katman tipleri

*[4]

Polyvinyl butyral (PVB)



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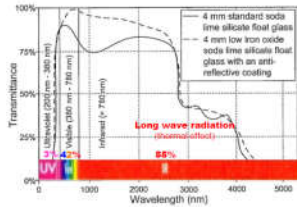
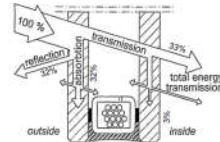
1-MALZEME VE ÜRÜN ÖZELLİKLERİ

Isı Cam

*[4]

Glass has very high transparency within the visible range of wavelengths ($\lambda = 380-750$ nm).

Total solar radiation reaching the outer glass pane



Most energy from solar radiation is contained in the IR long wave radiation (55%). Therefore, the strategy for solar protection is to block as much IR as possible without reducing the transmittance in the visible spectrum.

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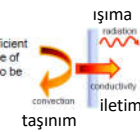
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1-MALZEME VE ÜRÜN ÖZELLİKLERİ

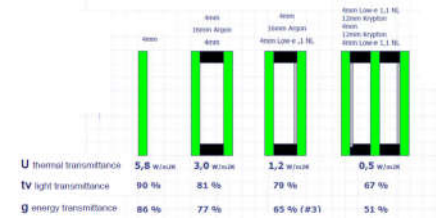
Isı Cam

*[4]

In order to reduce coefficient U the thermal resistance of the glass element has to be increased.



It is not possible to change the convection properties but conductivity can be reduced by adding air space elements (preferably with a heavy gas: lower thermal conductivity) and heat transfer by radiation can be reduced by low emissivity coatings.



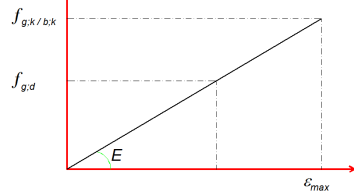
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HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Dayanım



Avrupa prEN13474-1/2 1999 Design of Glass Panes
 Almanya TRLV:1998, TRAV:2003 ve DIN18008:2011
 USA ASTM E 1300 / GANA 2004
 Avustralya AS1288
 İngiltere BS 6262-3:2005

$f_{g,d}$: etkin emniyet gerilmesi (allowable effective stress for the design)
 $f_{g,k/b,k}$: karakteristik test dayanımı (characteristic value of the test strength)
 E : elastisite modülü (modulus of elasticity) =7000kN/cm²
 G : kayma modülü (shear modulus) =2900 kN/cm²
 ν : poisson oranı (Poisson's Ratio) =0,20
 α : ısı genişleme katsayısı (coef. of linear th. exp.) =0,000012 1/C°
 ρ : birim kütle (unit mass) =2500 kg/m³

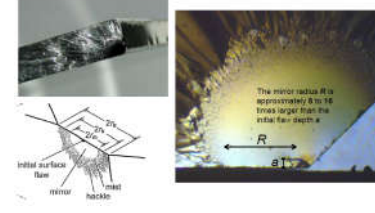
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2-TEMEL TASARIM İLKELERİ

Dayanım

*[4]

Camın teorik çekme gerilmesi 5000~8000MPa arasındadır. Ancak yüzey kusurları sebebiyle gerçek dayanımı çok daha düşüktür. Cam sünek bir malzeme olmadığı için yüzeyindeki üretim ve işleme sebebiyle mevcut çatlaklardaki büyük gerilme yığılmaları tekrar dağılıma uğrayamaz. Bu sebeple tavlanmış camın eğilme dayanımı 30~80MPa arasındadır



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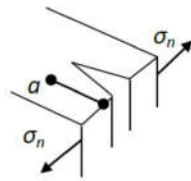
2-TEMEL TASARIM İLKELERİ

Dayanım

*[4]

There is stress magnification near the tip of a crack.

The stress intensity factor K_I : elastic stress intensity near a crack tip. Provides a means to characterize the material in terms of its fracture toughness.



$$K_I = Y \cdot \sigma_n \cdot \sqrt{\pi \cdot a}$$

K_I : stress intensity factor [MPa.m^{1/2}]
 Y : geometry factor [-]
 σ_n : stress normal to the flaw's plane [MPa]
 a : flaw depth [m]

Instantaneous failure of glass occurs when the elastic stress intensity K_I , due to tensile stress at the tip of a crack, reaches or exceeds a critical value. This critical value is a material constant known as the fracture toughness or the critical stress intensity factor K_{Ic} .

HESAP ESASLARI

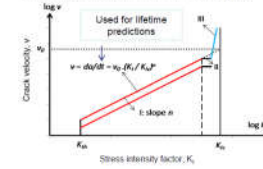
2-TEMEL TASARIM İLKELERİ

Dayanım

*[4]

The growth of a surface flaw depends on the properties of the flaw and the glass, the stress history and the relationship between crack velocity and stress intensity.

The crack velocity scales with the kinetics of the chemical equation for the stress corrosion (region I).



R, v_0 : crack velocity parameters for structural design no.10 is reasonable and $v_0=50\text{mm}^3/\text{s}$ should be conservative

K_{Ic} - Fracture toughness (material constant = 0.75 MPa m^{1/2} for SLSG)

K_{Ic} - Threshold below which no crack growth occurs = 0.55 MPa m^{1/2} for SLSG

In region III, close to K_{Ic} , v is independent of the environment and approaches a characteristic propagation speed very rapidly ($\approx 1500\text{m/s}$).

In region II the kinetics of the chemical reaction at the crack tip are no longer controlled by the activation of the chemical process but by the supply rate of water (water rate can't keep up when the crack speed increases very fast)

For usual conditions, only region I (extremely slow sub-critical crack growth) is relevant for determining the design life of a glass element.

Parameters affecting the relation between v and stress intensity factor K_I :

Humidity, temperature, PH value.
 Loading rate (if it is too fast the water supply suffer a shortage and the stress corrosion is slow dv/dt).
 Chemical composition of glass (affects all the parameters in sub critical crack growth).

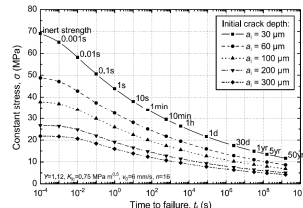
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2-TEMEL TASARIM İLKELERİ

Dayanım

Cam dayanımı birçok etkene bağlıdır: *[4,6,7]

- yüzeysel durumu (çatlakların boyutları)
- imalat yüzeyi (alt, üst): erimiş kalay ve hava tarafındaki yüzeyler üretildiğinde farklı dayanım gösterir ancak zamanla bu fark azalır.
- yüzeysel alanı
- yüklemeye süresi
- çevresel koşullar



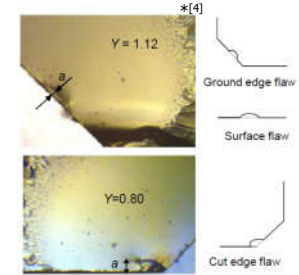
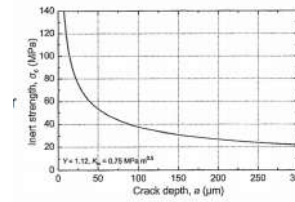
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2-TEMEL TASARIM İLKELERİ

Dayanım



$$f_{inert} = \frac{0.75}{Y \sqrt{\pi a_c}} \text{ MPa}$$

Stress causing failure of a crack of depth a_c (a_c , critical flaw depth)
Resistance of a crack to instantaneous failure (not triggered by sub critical crack growth)

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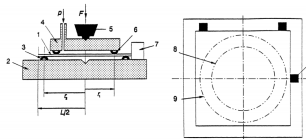
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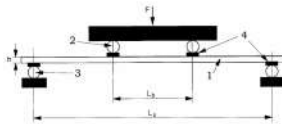
2-TEMEL TASARIM İLKELERİ

Dayanım Testleri

EN1288 *[8]



Çift halka deneyi



Dört nokta eğilmesi deneyi

- 1 Deneysel numunesi $L_0 = (200 \pm 1) \text{ mm}$
- 2 Eğme silindiri
- 3 Destek silindiri $L_1 = (11000 \pm 2) \text{ mm}$
- 4 Kauçuk yastıklar

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2-TEMEL TASARIM İLKELERİ

Tasarımda Dikkate Alınan Dayanım

*[5]

$$f_{R,d} = k_{mod} \frac{f_{R,k}}{\gamma_m k_A} \gamma_n$$

prEN13474

Table 6: Modification factor k_{mod}

Duration of action	Loading example	k_{mod}
short	wind	0.72
medium	snow	0.95
permanent	climate (for insulating glass units)	0.95
	altitude (for insulating glass units)	0.97

Glass product	Type	Thermally tempered heat-treated safety		Chemically strengthened	
		Ultimate limit state	Serviceability limit state	Ultimate limit state	Serviceability limit state
Flat glass	1.6	1.0	2.3	1.5	2.3
	2.3	1.5	2.3	1.5	2.3

Glass type	Processing	Uniformly distributed load duration					
		Short duration loads		Medium duration loads		Permanent loads	
Ultimate limit state	Serviceability limit state	Ultimate limit state	Serviceability limit state	Ultimate limit state	Serviceability limit state	Ultimate limit state	Serviceability limit state
Heat processed flat glass	annealed	17.0	30.7	8.5	15.5	6.1	11.5
	tempered	27.9	42.3	19.6	32.0	17.3	28.2
Heat processed insulating glass	annealed	49.6	80.7	41.1	65.3	39.0	61.5
	tempered	62.7	100.7	54.2	85.3	52.0	81.5

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HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Tasarımda Dikkate Alınan Dayanım

Design standard: Calculation of R_d according to prEN 16612 [37] and prEN 523-002 [49]	
Prestressed glass	Annealed glass
$R_d = \frac{k_{mod} \cdot k_{sp} \cdot f_{g,p}}{\gamma_{M,2}} \cdot k_T \cdot (f_{k,p} - f_{g,p})$	$R_d = \frac{k_{mod} \cdot k_{sp} \cdot f_{g,p}}{\gamma_{M,2}}$
Material partial factor	
$\gamma_{M,2} = 1.2$	$\gamma_{M,2} = 1.8$
Strength	
$f_{g,p}$: Characteristic value of the bending strength of annealed glass	
$f_{k,p}$: bending strength according to the product standard of prestressed glass	
Factor of load duration $k_{mod} = 0.663t^{-0.19}$	
t : load duration in hours; $k_{mod,max} = 0.25$, $k_{mod,min} = 1$. Definition of the load duration: see Code Review No. 2.	
k_{sp} : Strengthening factor of prestressed glass (depending on the manufacturing process), 1.0 for horizontal toughening, 0.6 for vertical toughening	
k_{sp} : factor for the glass surface profile, e.g. 1.0 for float glass and 0.75 for patterned glass	

*[9]

prEN16612

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2-TEMEL TASARIM İLKELERİ

Tasarımda Dikkate Alınan Dayanım

Eurocode Outlook No. 22

- (1) The Eurocode should take into account different load combinations for different classes of structural glazing. Special Consequences classes for glass should be specified, further differentiating those of EN 1990, i.e. the indicated classes do not comply with those of the current EN 1990.
- (2) A classification can be:
 - CC0: Elements only responsible for its own stability, no personal loading. There are low consequences when the element fails.
 - CC1: Elements only responsible for its own stability, personal loading. There are rather low consequences when the element fails.
 - CC2: Primary elements or elements only responsible for its own stability, personal loading. There are medium consequences when the element fails.
 - CC3: Primary elements. There are serious consequences when the element fails.
- (3) The Eurocode should establish a model to predict the consequences of a glass failure and to determine the accidental scenario.

*[10]

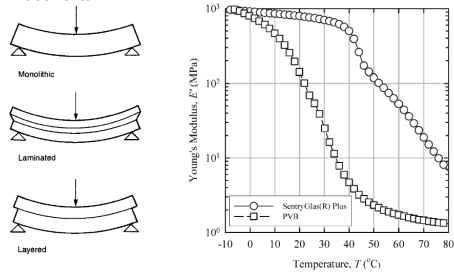
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2-TEMEL TASARIM İLKELERİ

Ara katmanlı camlar – lamine camlar

*[11]

Lamine camlarda ara katman olarak en çok kullanılan PVB viskoelastik bir malzemedir. Kayma modülü sıcaklık ve yükün etkime süresine göre farklılık gösterir. Kayma modülü ne kadar yüksekse iki camın kompozit çalışma etkisi o kadar artar.



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2-TEMEL TASARIM İLKELERİ

Ara katmanlı camlar – lamine camlar

*[10]

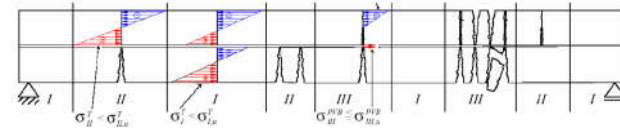


Figure 5-4 Resistance mechanisms in the post glass breakage phase [178]

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2-TEMEL TASARIM İLKELERİ

Yükler

Cephe sistemlerinin direnç göstermesi gereken yükler aşağıda özetlenmiştir (EN 13830):

*[12]

- zati yükler (EN 1991-1-1)
- hareketli yükler (EN 1991-1-1)
- kar yükleri, (EN 1991-1-3)
- rüzgar yükleri (EN 1991-1-4)
- deprem yükleri (EN 1998-1, TDY Denk 2.21)
- çarpma yükleri (EN 12600)
- bina hareketleri ve ısı yükler
- diğer yükler (patlama, termal gerilmeler)

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2-TEMEL TASARIM İLKELERİ

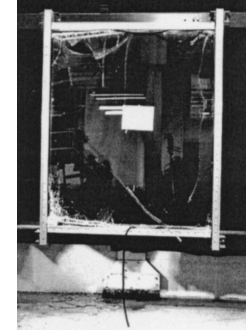
Yükler

DEPREM YÜKLERİ

Mimari camların yeterli bir deprem performansı göstermesini hedeflemenin iki önemli sebebi:

- İnsanların düşen camlardan dolayı yaralanmasını veya can kayıplarını önlemek
- Binanın servis dışı kalmasından ve tamirattan dolayı oluşacak maliyeti düşürmek

*[13]



HESAP ESASLARI

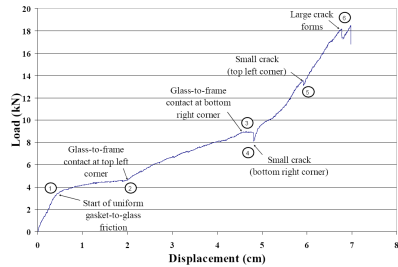
2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

*[14]

- 0.01 cm/s statik yük altında yük-deplasman ilişkisi



HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

Çerçevesi cam giydirmeye cephelerde, cam düzlemine dik (rüzgar yükü gibi) deprem yüklerinin yanı sıra -kat ötelenmesinden kaynaklanan- cam düzlemi içinde etkiler de oluşur.

- Cephelerde düzlem içi ve düzlem dışı deprem yükleri

$$f_e = 0.5 A_0 I w_e \left(1 + 2 \frac{H_i}{H_N} \right)$$

*[15] Denklem 2.21

HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

ASCE7-05'e göre [16]

13.5.9 Glass in Glazed Curtain Walls, Glazed Storefronts, and Glazed Partitions.

13.5.9.1 General. Glass in glazed curtain walls, glazed storefronts, and glazed partitions shall meet the relative displacement requirement of Eq. 13.5-1:

$$\Delta_{fallout} \geq 1.25D_p \quad (13.5-1)$$

or 0.5 in. (13 mm), whichever is greater where:

where

$\Delta_{fallout}$ = the relative seismic displacement (drift) at which glass fallout from the curtain wall, storefront wall, or partition occurs (Section 13.5.9.2)

D_p = the relative seismic displacement that the component must be designed to accommodate (Eq. 13.3-2). D_p shall be applied over the height of the glass component under consideration

I = the occupancy importance factor (Table 11.5-1)

EXCEPTIONS:

$$D_{clear} \geq 1.25D_p$$

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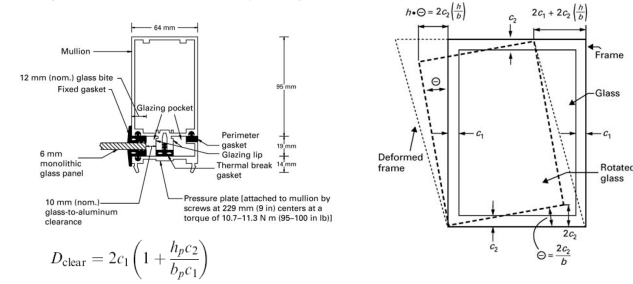
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Yükler

DEPREM YÜKLERİ

- Cephelerde düzlem içi deplasman şartları:



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*[17]

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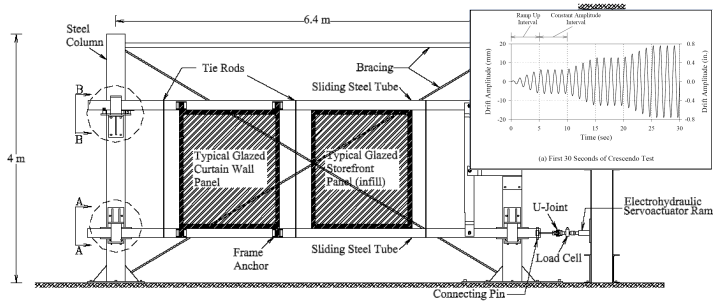
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2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

(AAMA 501.6-09) [18]



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2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

%2.50 drift

GÖRSEL

DEPREM DENEYİ-1



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2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

GÖRSEL

DEPREM DENEYİ-2

HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Yükler

DEPREM YÜKLERİ

GÖRSEL

DEPREM DENEYİ-3

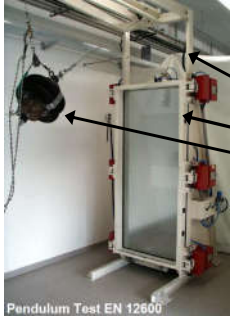
HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Yükler

ÇARPMA YÜKLERİ (TS EN 12600'e göre)

[19]



Pendulum Test EN 12600

- 1 Ana çerçeve
- 2 Tuturma çerçevesi
- 3 Darbe uygulayıcı

Sınıf	Düşme yüksekliği, mm
3	190
2	450
1	1200

HESAP ESASLARI

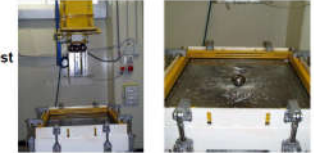
2-TEMEL TASARIM İLKELERİ

Yükler

DARBE YÜKLERİ (TS EN 356'ya göre)

[20]

Ball drop impact test
EN 356



Axe & Hammer Test
EN 356



HESAP ESASLARI

2-TEMEL TASARIM İLKELERİ

Performans Kriterleri
EN 13830

*[12]

5.7 Wind load resistance

The curtain walling kit shall be tested in accordance with EN 12179. Under the imposed wind loads only the maximum frontal deflection (δ) of the curtain walling's framing members shall not exceed the following limits:

- $d \leq L/200$, if $L \leq 3000$ mm;
- $d \leq 5$ mm + $L/300$, if 3000 mm $< L < 7500$ mm;
- $d \leq L/250$, if $L \geq 7500$ mm.

when measured between the points of support or anchorage to the building's structure (L).

In addition, the permissible deflection limits of the infill (e.g. IGU, stone, etc.) shall be taken into account.

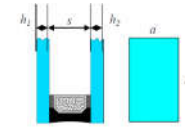
NOTE Guidance on the combinations of different loads is given in Annex C.

HESAP ESASLARI

3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar

In case of double glazing, with panes of thickness h_1 and h_2 , the distribution of external uniformly distributed loads (e.g. wind, snow, self weight) is essentially determined by the distribution of the stiffness of the panes, that is:



$$\delta_1 = \frac{h_1^3}{h_1^3 + h_2^3}$$

$$\delta_2 = \frac{h_2^3}{h_1^3 + h_2^3} = 1 - \delta_1$$

Additionally, the distribution of external loads is determined by the insulating unit factor ϕ .

$$\phi = \frac{1}{1 + (a/s)^4}$$

The length "a" gives the actual dimension of the unit (e.g. in a rectangular unit the length of the short edge) while "a*" is the characteristic length of the unit, depending on the thickness of the glass panes (h_1 and h_2), the gas space (s), and the shape of the unit (ν).

$$a^* = 28,9 \cdot \left(\frac{s \cdot h_1^3 \cdot h_2^3}{(h_1^3 + h_2^3) \cdot k_s} \right)^{0,25}$$

HESAP ESASLARI

3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar

[5]

Coefficient k_s for calculation of the volume change

Linear interpolation apply.

For small deflections (linear theory) $p=0$ may be considered.

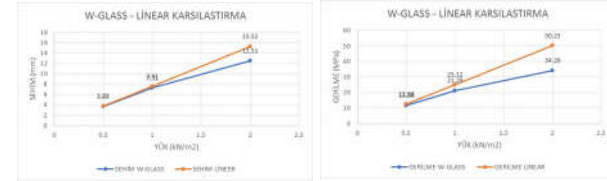
L4 Insulated glass units

A/B	p* (internal initial pressurization)									
	0	5	10	20	30	50	100	200	300	500
1.0	0.019	0.019	0.019	0.018	0.017	0.015	0.011	0.008	0.007	0.005
0.9	0.024	0.024	0.023	0.022	0.020	0.017	0.013	0.009	0.007	0.005
0.8	0.029	0.029	0.028	0.026	0.023	0.020	0.015	0.010	0.008	0.007
0.7	0.035	0.035	0.034	0.031	0.028	0.023	0.017	0.012	0.010	0.008
0.6	0.042	0.042	0.040	0.037	0.033	0.027	0.020	0.014	0.012	0.009
0.5	0.050	0.050	0.048	0.044	0.040	0.033	0.025	0.018	0.014	0.011
0.4	0.058	0.058	0.055	0.051	0.046	0.040	0.031	0.022	0.018	0.014
0.3	0.068	0.067	0.064	0.059	0.054	0.046	0.035	0.025	0.020	0.015
0.2	0.077	0.077	0.074	0.070	0.064	0.054	0.042	0.030	0.024	0.018
0.1	0.086	0.086	0.083	0.078	0.072	0.061	0.048	0.034	0.027	0.020
0	0.095	0.095	0.092	0.086	0.079	0.066	0.051	0.036	0.029	0.021

HESAP ESASLARI

3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar

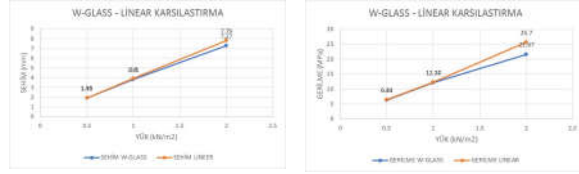


PROBLEM	GEOMETRİ	A(mm)	B(mm)	Ap(°)	Dış cam (mm)	Hava boşluğu (mm)	İç cam (mm)	Yük (kN/m²)	Gerilme hesabı için yük payı	W-GLASS		LINEER		Sehim fark	Gerilme fark
										Sehim (mm)	Gerilme (Mpa)	Sehim (mm)	Gerilme (Mpa)		
B-1	DIKDÖRTGEN	2000	1000	90	6	-	-	0.5	1.5	3.72	11.88	3.81	12.56	-0.02	-0.06
B-2	DIKDÖRTGEN	2000	1000	90	6	-	-	1	1.5	7.31	21.29	7.61	25.12	-0.04	-0.18
B-3	DIKDÖRTGEN	2000	1000	90	6	-	-	2	1.5	12.53	34.28	15.32	50.23	-0.22	-0.47

HESAP ESASLARI

3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar



PROBLEM	GEOMETRİ	A(mm)	B(mm)	Açı(°)	Dış cam (mm)	Hava boşluğu (mm)	İç cam (mm)	Yük (kN/m²)	Gerilme hesabı için yük çarpanı	W-GLASS		LINEAR		Sehim fark	Gerilme fark
										Sehim (mm)	Gerilme (Mpa)	Sehim (mm)	Gerilme (Mpa)		
B-1	DIK DÖRTGEN	2000	1000	90	6	16	6	0.5	1.5	1.93	6.31	1.95	6.43	-0.01	-0.02
B-2	DIK DÖRTGEN	2000	1000	90	6	16	6	1	1.5	3.81	12.14	3.9	12.27	-0.02	-0.01
B-3	DIK DÖRTGEN	2000	1000	90	6	16	6	2	1.5	7.27	21.67	7.79	25.7	-0.07	-0.19

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3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar

[4,5]

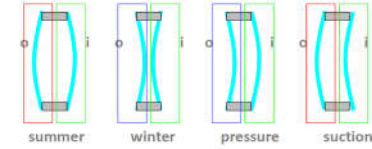
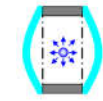
Internal pressure transmits the external loads (e.g. wind on pane 1) from one pane to the next ("Coupling Effect")

External load	Part of the external load carried by pane 1	Part of the external load carried by pane 2
F_{e1} acting on pane 1	$(1-\varphi) \cdot \delta_1 \cdot F_{e1}$	$(\delta_1 + \varphi \cdot \delta_2) \cdot F_{e1}$
F_{e2} acting on pane 2	$(1-\varphi) \cdot \delta_2 \cdot F_{e2}$	$(\varphi \cdot \delta_1 + \delta_2) \cdot F_{e2}$

The internal loads, given by the isochore pressure, are reduced by a factor proportional to the relative flexibility of the panes.

Internal load	Part of the internal load carried by pane 1	Part of the internal load carried by pane 2
Isochore pressure Dp	$\pm \varphi \cdot \Delta p$	$\pm \varphi \cdot \Delta p$

"Climatic Load"



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3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

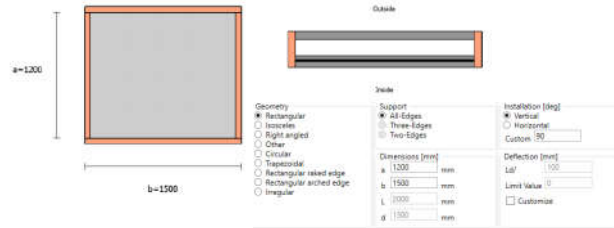
Isı Camlar - Örnek

Project Name / No :
Revision :
Date : 02-May-2017

Design Report



GEOMETRY / SECTION



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3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar - Örnek

SYSTEM-INFO

DESIGN CODE : prEN 13474
GEOMETRY : Rectangular
PANE COUNT : 2
SUPPORT : All-Edges
DIMENSION (a) : 1200 (mm)
DIMENSION (b) : 1500 (mm)
SLOPE ANGLE: 90 (deg) VERTICAL
DEFLECTION LIMIT : L(smallest)/100

Actions	(High: >400 meters)
Low Altitude	
High Altitude	
Altitude Summer	3.6 kN/m2
Altitude Winter	-3.6 kN/m2
Climate Summer	1.2 kN/m2
Climate Winter	-1.5 kN/m2
Wind Pressure	2.5 kN/m2
Wind Suction	-2.5 kN/m2
Snow	0 kN/m2

SECTION INFO (from Outside to Inside)			
Thickness [mm]	Type	Material	Prestressing
8	Glass	Float Glass	Thermally Toughened
16	Cavity		
4	Glass	Float Glass	NonPrestressed Annealed
1.52	PVB		
6	Glass	Float Glass	NonPrestressed Annealed

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HESAP ESASLARI

3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar - Örnek

Name	DL	ALS	ALW	CLS	CLW	SL	WLP	WLS
E1_ULS	1.35	0	0	0	0	0	0	0
E3_ULS	1.35	0	0	0	0	1.5	0	0
E5_ULS_a1	1.35	0	0	0	0	0	1.5	0
E5_ULS_a2	1.35	0	0	0	0	0	0	1.5
E5_ULS_a3	1.35	0	0	0	0	1.35	1.35	0
E5_ULS_a4	1.35	0	0	0	0	1.35	0	1.35
E1_SLS	1	0	0	0	0	0	0	0
E2_SLS(+)	1	1	0	0	0	0	0	0
E2_SLS(-)	1	-1	0	0	0	0	0	0
E3_SLS	1	0	0	0	0	1	0	0
E4_SLS_1(+)	1	1	0	0.9	0	0.9	0	0
E4_SLS_1(-)	1	-1	0	0.9	0	0.9	0	0
E4_SLS_2(+)	1	1	0	0	0.9	0.9	0	0
E4_SLS_2(-)	1	-1	0	0	0.9	0.9	0	0
E5_SLS_a1	1	0	0	0	0	0	1	0
E5_SLS_a2	1	0	0	0	0	0	0	1
E5_SLS_a3	1	0	0	0	0	0.9	0.9	0
E5_SLS_a4	1	0	0	0	0	0.9	0	0.9
E5_SLS_b1	1	1	0	0.9	0	0.9	0.9	0
E5_SLS_b2	1	1	0	0	0.9	0.9	0.9	0
E5_SLS_b3	1	1	0	0.9	0	0.9	0	0.9
E5_SLS_b4	1	1	0	0	0.9	0.9	0	0.9
E5_SLS_b1(+)	1	-1	0	0.9	0	0.9	0.9	0
E5_SLS_b2(+)	1	-1	0	0	0.9	0.9	0.9	0
E5_SLS_b3(+)	1	-1	0	0.9	0	0.9	0	0.9
E5_SLS_b4(+)	1	-1	0	0	0.9	0.9	0	0.9

prEN 13474

DL:Dead Load
SL:Snow Load
ALS:Altitude Summer
ALW:Altitude Winter
CLS:Climate Summer
CLW:Climate Winter
WLP:Wind Load Pressure
WLS:Wind Load Suction
SLS:Serviceability Limit State
ULS:Ultimate Limit State

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HESAP ESASLARI

3-İKİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

Isı Camlar - Örnek

DECISION: Glass unit is SUFFICIENT (Max utilized capacity %71)

Outer Pane max Stress Utilization (%23) : OK
Outer Pane max Displacement Utilization (%55) : OK
Inner Pane max Stress Utilization (%71) : OK
Inner Pane max Displacement Utilization (%39) : OK

macrostatic

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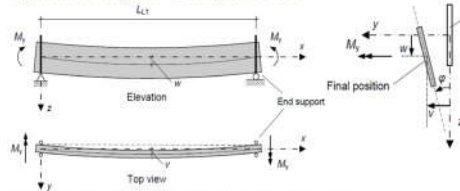
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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

KİRİŞLER

A) Monolithic glass – Analytical model



$$M_{CR} = C_1 \frac{\pi^2 EI_z}{L^2} \left[\sqrt{(C_2 z_0)^2 + \frac{GI_t E^2}{\pi^2 EI_z}} + C_2 z_0 \right] \text{ Critical LTB Moment}$$

- C_1, z_0, \dots take into account different boundary conditions, different bending moment, distance between the centre of gravity and the load point
- LTB formulas for steel are valid, e.g. EC3

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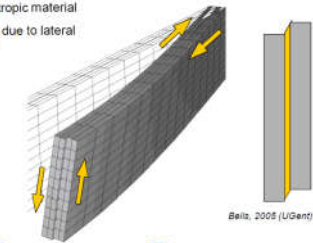
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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI

KİRİŞLER

B) Laminated glass – Analytical model

- No homogeneous, isotropic material
- Shear deformation due to lateral bending & torsion



$$M_{CR} = C_1 \frac{\pi^2 E_g I_z}{L^2} \left[\sqrt{(C_2 z_0)^2 + \frac{(GI_t)_{eff}^2}{E_g I_z}} + C_2 z_0 \right] \text{ Critical LTB Moment}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI HİBRİT KİRİŞLER [4]

Möhler's method for steel-glass beam

Effective moment of inertia

$$I_{y,eff} = 2 \cdot I_a + \underbrace{\left(\frac{E_s}{E_g} \right)}_{\text{stiffness ratio}} I_s + 2 \cdot \underbrace{\left(\gamma \cdot A_s \cdot z_a^2 \right)}_{\text{efficiency factor}}$$

$$\gamma = \frac{1}{1+k} \quad k = \pi^2 \cdot \frac{E_s \cdot A_s}{L^2 \cdot K_K}$$

$$K_K = G_K \cdot \frac{b_k}{I_k}$$

Stiffness of glued connection:
shear modulus of adhesive

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI HİBRİT KİRİŞLER [4]

Möhler's method for steel-glass beam

Evaluation of stresses

$$\sigma_{1,2} = \pm \frac{M_y}{I_{y,eff}} \cdot \left(\gamma \cdot z_a + \frac{t_f}{2} \right)$$

$$\sigma_{1,2} = \pm \frac{M_y}{I_{y,eff}} \cdot \frac{h_g}{2} \cdot \frac{E_s}{E_g}$$

Vertical deflection of the beam

$$\delta = \frac{5}{384} \cdot \frac{q \cdot L^4}{E_g \cdot I_{y,eff}}$$

Shear stress in glued connection

$$\tau = \frac{Q \cdot \gamma \cdot z_a \cdot A_s}{I_{y,eff} \cdot b_k}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI HİBRİT KİRİŞLER

L _{sg} =	9000 mm	frodaj	40 Mpa
h=	400 mm	Gk	1.6 Mpa
b=	30 mm	bk=	30 mm
bsteel=	30 mm	ik=	10 mm
hsteel=	50 mm	KK=	4.8 Mpa
hcsm=	280 mm	Ea=	200000 MPa
ly,eff=	625000	19208000	17628244.1
	flanges	web	flanges
ly,eff=	37461244		8
ly,glass=	160000000		0.116071
%	0.23413278	WL	0.8 kNm2
MRd,eff=	30.5806074	b	4.5 m
MRd,glass	32	A _{max}	46.91
L _{sg} =	9000 mm	frodaj	40 Mpa
h=	400 mm	Gk	1.6 Mpa
b=	30 mm	bk=	30 mm
bsteel=	80 mm	ik=	10 mm
hsteel=	20 mm	KK=	4.8 Mpa
hcsm=	340 mm	Eg=	200000 MPa
ly,eff=	106666.667	34391000	34877391.1
	flanges	web	flanges
ly,eff=	69375058		3
ly,glass=	160000000		0.247147
%	0.43359411	WL	0.8 kNm2
MRd,eff=	46.6386943	b	4.5 m
MRd,glass	32	A _{max}	25.33

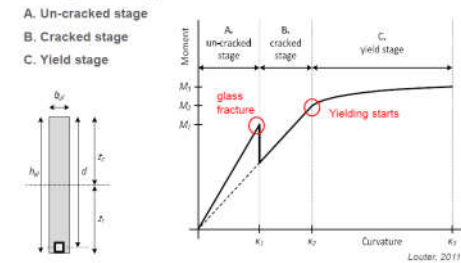
Hibrit kirişlerde çelik başlıkların gövdenin geniş olması durumu eğilme kapasitesi ve rijitliği arttırmaktadır.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI HİBRİT KİRİŞLER [4]

Analytical model of reinforced glass beams

- Analogy with reinforced concrete
- Full interaction between glass and reinforcement is assumed.
- Three stages are distinguished:



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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI HİBRİT KİRİŞLER [4]



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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.1 Material properties

Material Property	Value
Characteristic bending strength of basic annealed glass $f_{g,k}$	45N/mm ²
Density ρ	2500kg/m ³
Young's modulus E	70000N/mm ²
Shear modulus G	28700N/mm ²
Poisson's ratio μ	0.22

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Characteristic strength of toughened glass

$$f_{gd} = \frac{k_{mod} k_{sp} f_{gk}}{\gamma_{M,A}} + \frac{k_v (f_{bk} - f_{gk})}{\gamma_{M,v}} \quad \dots \text{Equation C.1}$$

where:

- f_{gk} is the characteristic strength of basic annealed glass (45N/mm²)
- k_{mod} is the factor for load duration
- k_{sp} is the factor for glass surface profile
- $\gamma_{M,A}$ is the material partial factor for basic annealed glass
- k_v is the factor derived from the method of strengthening of the glass
- f_{bk} is the characteristic bending strength of prestressed glass
- $\gamma_{M,v}$ is the material partial factor for surface prestressed glass

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.2 Material partial factors

Type of glass	Material partial factor
Basic annealed glass	$\gamma_{M,A} = 1.6^a$
Surface prestress	$\gamma_{M,v} = 1.2$

Note

a The material partial factor of 1.6 stated does not correlate with that given in Table 2 of prEN 16612, which indicates a value of 1.8. This is because once a formal European code of practice is adopted, a UK National Annex will likely be created alongside it that will recommend a value of 1.6 be used for the material partial factor for basic annealed glass. For all other regions the default value is 1.8, unless advised otherwise by respective national annexes.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.3 Variable action partial factor γ_0

Type of element	Partial factor for variable actions
Primary structure	1.5
Secondary structure	1.3
Infill panel	1.2
Low risk infill panel ^a	1.1

Note
a An infill panel whose failure would not cause injury.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.4 Glass surface profile factor k_{gp}

Type of glass	As produced ^a	Sandblasted
Float glass	1.0	0.6
Drawn sheet glass	1.0	0.6
Enamelled float or drawn sheet glass	1.0	0.6
Patterned glass	0.75	0.45
Enamelled patterned glass	0.75	0.45
Polished wired glass	0.75	0.45
Patterned wired glass	0.6	0.36

Note
a Where glass is etched with acid the 'as produced' value of k_{gp} should be used.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Load duration factor k_{mod}

The base equation to determine the value of the load duration factor k_{mod} is as shown in Equation C.2.

$$k_{mod} = 0.663t^{-1/16} \quad \dots \text{Equation C.2}$$

where:
t is the duration in hours

The equation can only be used for load durations that are more than 20msec.

Laminated glass effective thickness $t_{ef,w}$ and $t_{ef,v}$
The effective thickness of laminated glass in terms of bending stress, $t_{ef,w}$, is defined in Equation C.3.

$$t_{ef,w} = \sqrt[3]{\sum_k h_k^3 + 12\alpha \left(\sum_k t_k \eta_{k,i}^2 \right)} \quad \dots \text{Equation C.3}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.5 Action duration factor k_{red}

Duration	Example	k_{red}
5 seconds	Single gust	1.00
30 seconds	Domestic balustrade load	0.89
5 minutes	Workplace/public balustrade load	0.77
10 minutes	Multiple gust (storm)	0.74
30 minutes	Maintenance access	0.69
5 hours	Pedestrian access	0.60
1 week	Snow load short-term	0.48
1 month	Snow load medium-term	0.44
3 months	Snow load long-term	0.41
50 years	Permanent (e.g. self-weight and altitude pressure)	0.29

Note
a The minimum value of k_{red} is 0.25.
b In the case of extremely short loading conditions such as explosions, k_{red} can be greater than 1.0.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.6 Characteristic bending strength for prestressed glass $f_{0,k}$

Base glass material	Characteristic bending strength $f_{0,k}$ (N/mm ²)		
	Thermally toughened	Heat-strengthened	Chemically toughened
Float glass or drawn sheet	120	70	150
Patterned	90	55	100
Enamelled float glass or drawn sheet ^a	75	45	
Enamelled patterned glass ^a	75	45	

Note
a Enamelled glass cannot be chemically toughened.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

The effective thickness of laminated glass in terms of stress within a ply, $h_{\text{eff},i}$, is defined in Equation C.4.

$$h_{\text{eff},i} = \sqrt{\frac{h_{i,i}^3}{h_1 + 2\omega h_{m1}}} \quad \dots \text{Equation C.4}$$

where:

ω is the coefficient of shear transfer of the interlayer, which varies from 0 to 1.
 h_k and h_l are the thicknesses of the plies of glass within a laminated sheet, as defined in Figure C.1, i.e. $h_1 = h_k$ and $h_2 = h_l$
 h_{m1} is the distance to the mid-plane of the glass plies

The coefficient of shear transfer ω is based on the material of the interlayer and the duration of load the glass is to be subjected to. The different material types currently in use along with their stiffness family are as follows:
 - Acoustic PVB: Family 1.
 - Standard grade PVB: Family 2.
 - Ionoplast: Family 3.

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.8 Edge stress factor k_e

Glass type	Edge strength factor k_e		
	As-cut, arrisred, or ground edges ^a	Seamed edges ^b	Polished edges
Float or sheet glass	0.8	0.9	1.0
Patterned glass	0.8	0.8	0.8
Polished wired glass	0.8	0.8	0.8
Wired patterned glass	0.8	0.8	0.8

Note
 a Arrisred or ground edges by machined or hand where the abrasive action is across the edge.
 b Arrisred or ground edges by machine or hand where the abrasive action is along the length of the edge.

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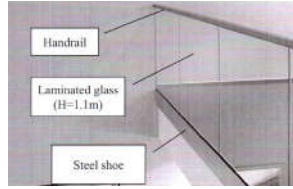
4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI TASARIM VERİLERİ [22]

Table C.9 Coefficient of shear transfer ω

Load case	Family 2	Family 3
Wind load (Mediterranean)	0.1	0.6
Wind load (other locations)	0.3	0.7
Personal load – normal duty	0.1	0.5
Personal load – crowds	0	0.3
Maintenance load	0	0.1
Snow loads – external canopies	0.1	0.3
Snow loads – roof	0	0.1
Permanent loads	0	0

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KORKULUK



Loading

$$q := 1 \text{ kN} \quad \gamma_F := 1.5$$

$$h := 1.1 \text{ m} \quad M := q \cdot h \cdot \gamma_F = 1.65 \text{ kN} \cdot \text{m}$$

Glass strength

$$f_{gk} := 45 \text{ MPa} \quad f_{bk} := 120 \text{ MPa} \quad E := 70 \text{ GPa}$$

$$k_{mod} := 0.89 \quad k_{sp} := 1.0 \quad k_v := 1$$

$$\gamma_{MA} := 1.8 \quad \gamma_{Mv} := 1.2$$

$$f_{g,d} := \left[\frac{k_{mod} \cdot k_{sp} \cdot f_{gk}}{\gamma_{MA}} + \frac{k_v \cdot (f_{bk} - f_{gk})}{\gamma_{Mv}} \right] = 84.75 \text{ MPa}$$

HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KORKULUK

Let's assume 2x10mm fully toughened glass (PVB - family 2)

$$h_1 := 10 \text{ mm} \quad h_2 := h_1 \quad \omega := 0.1 \quad (\text{personal load - normal duty})$$

$$h_{eff,w} := \sqrt[3]{h_1^3 + h_2^3 + 12 \cdot \omega \cdot [2 \cdot h_1 \cdot (0.5 \cdot 1.52 \text{ mm} + 0.5 \cdot h_1)]^2} = 14.088 \text{ mm}$$

$$h_{eff,\sigma} := \sqrt{\frac{h_{eff,w}^3}{h_1 + 2 \cdot \omega \cdot (0.5 \cdot 1.52 \text{ mm} + 0.5 \cdot h_1)}} = 15.835 \text{ mm}$$

Stress check

$$W := \frac{1 \text{ m} \cdot h_{eff,\sigma}^2}{6} = 41.79 \text{ cm}^3 \quad \sigma := \frac{M}{W} = 39.483 \text{ MPa} < f_{g,d} = 84.75 \text{ MPa} \quad \text{OK}$$

Deflection check

$$l := \frac{1 \text{ m} \cdot h_{eff,w}^3}{12} = 23.302 \text{ cm}^4 \quad \delta := \frac{q \cdot h^3}{3 \cdot E \cdot I} = 27.2 \text{ mm} < \delta_{max} := \frac{h}{65} = 16.923 \text{ mm} \text{ NOT OK}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KORKULUK

Let's assume 2x12mm fully toughened glass (PVB - family 2)

$$h_1 := 12 \text{ mm} \quad h_2 := h_1 \quad \omega := 0.1 \quad (\text{personal load - normal duty})$$

$$h_{eff,w} := \sqrt[3]{h_1^3 + h_2^3 + 12 \cdot \omega \cdot [2 \cdot h_1 \cdot (0.5 \cdot 1.52 \text{ mm} + 0.5 \cdot h_1)]^2} = 16.836 \text{ mm}$$

$$h_{eff,\sigma} := \sqrt{\frac{h_{eff,w}^3}{h_1 + 2 \cdot \omega \cdot (0.5 \cdot 1.52 \text{ mm} + 0.5 \cdot h_1)}} = 18.905 \text{ mm}$$

Stress check

$$W := \frac{1 \text{ m} \cdot h_{eff,\sigma}^2}{6} = 63.254 \text{ cm}^3 \quad \sigma := \frac{M}{W} = 26.085 \text{ MPa} < f_{g,d} = 84.75 \text{ MPa} \quad \text{OK}$$

Deflection check

$$l := \frac{1 \text{ m} \cdot h_{eff,w}^3}{12} = 49.844 \text{ cm}^4 \quad \delta := \frac{q \cdot h^3}{3 \cdot E \cdot I} = 12.716 \text{ mm} < \delta_{max} := \frac{h}{65} = 16.923 \text{ mm} \text{ OK}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KORKULUK

Post-failure check (one ply broken)

$$\gamma_F := 1.0 \quad M := q \cdot h \cdot \gamma_F = 1.1 \text{ kN} \cdot \text{m}$$

$$h_1 := \begin{pmatrix} 8 \\ 10 \\ 12 \end{pmatrix} \text{ mm} \quad W := \frac{1 \text{ m} \cdot h_1^2}{6} \quad \frac{M}{W} = \begin{pmatrix} 103.125 \\ 66 \\ 45.833 \end{pmatrix} \text{ MPa} \quad f_{g,d} = 84.75 \text{ MPa}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KORKULUK

Pendulum test (simplified approach)
DIN 18008-4 Glass in Building – Design and construction rules – Part 4: Additional requirements for barrier glazing
Kategori B - drop height 700mm

$$\begin{aligned}
 m &:= 50\text{kg} && \text{mass} \\
 h &:= 70\text{cm} && \text{drop height} \\
 v &:= \sqrt{2g \cdot h} && v = 3.705 \frac{\text{m}}{\text{s}} \quad \text{max speed at impact} \\
 t_i &:= 15\text{ms} && \text{impact time} \\
 F_d &:= \frac{m \cdot v}{t_i} = 12.351 \cdot \text{kN} && \text{dynamic force} \\
 \frac{F_d}{20\text{cm} \cdot 20\text{cm}} &= 0.309 \cdot \text{MPa} && \frac{1.4 \cdot 120\text{MPa}}{1.0} = 168 \cdot \text{MPa}
 \end{aligned}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM DÖŞEME



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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM DÖŞEME

Loading (residential areas - floors) Panel 1500x1500mm

$$q := 2 \frac{\text{kN}}{\text{m}^2} \quad Q := 3\text{kN}$$

Glass strength (long-term)

$$f_{gk} := 45\text{MPa} \quad f_{bk} := 70\text{MPa} \quad E := 70\text{GPa}$$

$$k_{\text{mod}} := 0.29 \quad k_{\text{sp}} := 1.0 \quad k_v := 1$$

$$\gamma_{\text{MA}} := 1.8 \quad \gamma_{\text{Mv}} := 1.2$$

$$f_{g,d,\text{long}} := \left[\frac{k_{\text{mod}} \cdot k_{\text{sp}} \cdot f_{gk}}{\gamma_{\text{MA}}} + \frac{k_v \cdot (f_{bk} - f_{gk})}{\gamma_{\text{Mv}}} \right] = 28.083 \cdot \text{MPa}$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM DÖŞEME

Glass strength (short-term $t=5h$)

$$f_{gk} := 45\text{MPa} \quad f_{bk} := 70\text{MPa} \quad E := 70\text{GPa} \quad t := 5 \quad k_{\text{mod}} := 0.663 \cdot \frac{-1}{16}$$

$$k_{\text{mod}} := 0.6 \quad k_{\text{sp}} := 1.0 \quad k_v := 1$$

$$\gamma_{\text{MA}} := 1.8 \quad \gamma_{\text{Mv}} := 1.2$$

$$f_{g,d,\text{short}} := \left[\frac{k_{\text{mod}} \cdot k_{\text{sp}} \cdot f_{gk}}{\gamma_{\text{MA}}} + \frac{k_v \cdot (f_{bk} - f_{gk})}{\gamma_{\text{Mv}}} \right] = 35.822 \cdot \text{MPa}$$

Let's assume 3x12mm heat-strengthened glass (PVB - family 2)

$$h_1 := 12\text{mm} \quad h_2 := h_1 \quad h_3 := h_1$$

$$h_{\text{eff},w,\text{long}} := \sqrt[3]{h_1^3 + h_2^3 + h_3^3} = 17.307 \cdot \text{mm}$$

$$h_{\text{eff},\sigma,\text{long}} := \sqrt[3]{\frac{(h_1^3 + h_2^3)}{h_1}} = 16.971 \cdot \text{mm} \quad (\text{top ply as sacrificial})$$

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4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM DÖŞEME

$\omega := 0.1$ (personal load - normal duty)

$$h_{\text{eff},w,\text{short}} := \sqrt[3]{h_1^3 + h_2^3 + h_3^3 + 12 \cdot \omega \cdot [2 \cdot h_1 \cdot (0.5 \cdot h_2 + 1.52 \text{ mm} + 0.5 \cdot h_1)]^2} = 21.862 \text{ mm}$$

$$h_{\text{eff},\sigma,\text{short}} := \sqrt[3]{\frac{h_1^3 + h_2^3 + h_3^3 + 12 \cdot \omega \cdot [2 \cdot h_1 \cdot (0.5 \cdot 1.52 \text{ mm} + 0.5 \cdot h_1)]^2}{[h_1 + 2 \cdot \omega \cdot (0.5 \cdot 1.52 \text{ mm} + 0.5 \cdot h_1)]^2}} = 18.905 \text{ mm}$$

Stress check (based on Roark's Formulas for Stress and Strain)

1. Dead load (DL) $g := (3 \cdot b_1) \cdot 25 \frac{\text{kN}}{\text{m}^3} = 0.9 \text{ kPa}$

$a := 1500 \text{ mm}$ $b := a$

$\frac{a}{b} = 1$ $\alpha := 0.0444$ $\beta := 0.2874$

$\sigma_{DL} := \frac{1.35 \beta g b^2}{h_{\text{eff},\sigma,\text{long}}} = 2.728 \text{ MPa}$ $\delta_{DL} := \frac{\alpha g b^4}{E \cdot h_{\text{eff},w,\text{long}}} = 0.557 \text{ mm}$

HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM DÖŞEME

2. Uniformly distributed load (q) $q = 2.4 \text{ kPa}$

$a := 1500 \text{ mm}$ $b := a$

$\frac{a}{b} = 1$ $\alpha := 0.0444$ $\beta := 0.2874$

$\sigma_q := \frac{1.5 \beta q b^2}{h_{\text{eff},\sigma,\text{short}}} = 5.428 \text{ MPa}$ $\delta_q := \frac{\alpha q b^4}{E \cdot h_{\text{eff},w,\text{short}}} = 0.615 \text{ mm}$

3. Concentrated load (Q) $Q = 3.4 \text{ kN}$

$a := 1500 \text{ mm}$ $b := a$

$t_0 := 25 \text{ mm}$ $\rightarrow 0.5 \cdot h_{\text{eff},w,\text{short}} = 10.931 \text{ mm}$ $t_{0,1} := t_0$

$\alpha := 0.1267$ $\beta := 0.435$

$\sigma_Q := \frac{1.5 \beta Q}{2 \cdot \pi \cdot h_{\text{eff},\sigma,\text{short}}} \left[(1 + 0.22) \ln \left(\frac{2 \cdot b}{\pi \cdot t_{0,1}} \right) + \beta \right] = 29.332 \text{ MPa}$

$\delta_Q := \frac{\alpha Q b^2}{E \cdot h_{\text{eff},w,\text{short}}} = 1.169 \text{ mm}$

HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM DÖŞEME

Conditions:

$\frac{\sigma_{DL}}{f_{g,d,\text{long}}} + \frac{\sigma_q}{f_{g,d,\text{short}}} = 0.249$ $< .$ 1.0 **OK**

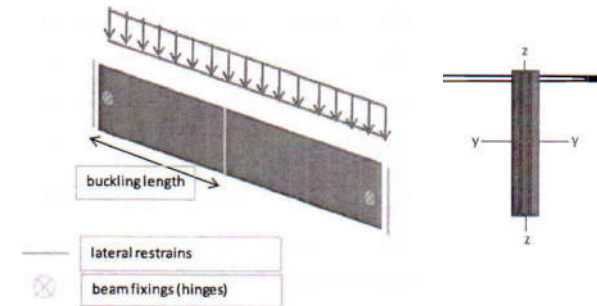
$\frac{\sigma_{DL}}{f_{g,d,\text{long}}} + \frac{\sigma_Q}{f_{g,d,\text{short}}} = 0.916$ $< .$ 1.0 **OK**

$\delta_{DL} + \delta_q = 1.172 \text{ mm}$ $< .$ $\frac{b}{100} = 15 \text{ mm}$ **OK**

$\delta_{DL} + \delta_Q = 1.727 \text{ mm}$ $< .$ $\frac{b}{100} = 15 \text{ mm}$ **OK**

HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KİRİŞ



HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KİRİŞ

The buckling verification is done using the Ayrton Parry method as per the Eurocodes. The imperfection factor used for calculating the normalised slenderness is from buckling curve c of EC3 according to chapter 5 of the book "Structural Use of Glass" - Mathias Haldimann, Andreas Luible, Mauro Overend that refers to studies and papers from Lindner and Holberndt and Luible.

$$\chi_{L11} = \frac{1}{\Phi_{L11} + \sqrt{\Phi_{L11}^2 - \lambda_{L11}^2}} \text{ but } \chi_{L11} \leq 1.0$$

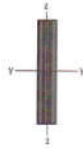
$$\text{where } \Phi_{L11} = 0.5 \left[1 + \alpha_{L11} (\chi_{L11} - 0.2) + \lambda_{L11}^2 \right]$$

α_{L11} is an imperfection factor

$$\lambda_{L11} = \sqrt{\frac{W_{L11}}{M_{cr}}}$$

M_{cr} is the elastic critical moment for lateral-torsional buckling

Properties around y-axis are calculated using the full thickness of the fin, properties around x-axis are calculated using the equivalent thickness



HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KİRİŞ

Length (m)	9	For heat treated fins the ratio between the height and the length shall typically be less than 15 (Eckelt does 20)
Buckling length (m)	9	
Spacing (m)	1.5	
Wind-load (Pa)	1500	
E (GPa)	70	
G (GPa)	28.5	$G = \frac{E}{2(1+\nu)}$
ν	0.23	
Allowable stress (MPa)	41.6	
Load safety factor	1.5	
Glass fin build-up (mm)	12	
Beam depth (mm)	500	

HESAP ESASLARI

4-BİRİNCİL CAM ELEMANLARIN BOYUTLANDIRILMASI ÇÖZÜMLÜ PROBLEMLER – CAM KİRİŞ

Equivalent thickness (mm)	35.93
Wy (cm ³)	2000.00
Wz (cm ³)	107.58
Iz (cm ⁴)	193.27
It (cm ⁴)	773.07
zg (mm)	250
C1	1.127
C2	0.454
Critical moment M_{cr} (Nm)	65790.1
Design moment M_{Ed} (Nm)	34171.9
Bending resistance $M_{y,Rd}$ (Nm)	83200.0
Normalised slenderness λ_{L11}	1.125
Imperfection factor α_{L11}	0.49
ϕ_{L11}	1.359
χ_{L11}	0.471
Bending+Torsional Buckling $M_{y,Rd}$ (Nm)	39216.0

$$M_{cr} = C_1 \frac{\pi^2 EI_z}{L^2} \sqrt{\frac{\lambda}{\lambda_1} + \frac{I^2 GI_z}{\pi^2 EI_z} + (C_2 z_g)^2} - C_2 z_g^2$$

No warping considered for solid rectangular sections

TEŞEKKÜRLER

- Sunuma yorum ve katkıları için CWG Danışmanlık'tan **Sn. Salih Sekban**'a teşekkürlerimizi sunuyoruz.
- Isıcamlar için paylaştıkları yazılım doğrulama örnek çözümleri sebebiyle MACROSTATIC'e teşekkür ederiz.

Kaynaklar

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- [8] TS EN 1288-1:2003 Cam - Yapılarda kullanılan – Eğilme Mukavemetinin Tayini - Bölüm 1: Camla ilgili deneylerin esasları

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- [19] TS EN 12600:2004 Cam - Yapılarda kullanılan – Sarkaç deneyi – Düz cam için çarpma deneyi ve sınıflandırma
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